

Fig. 1

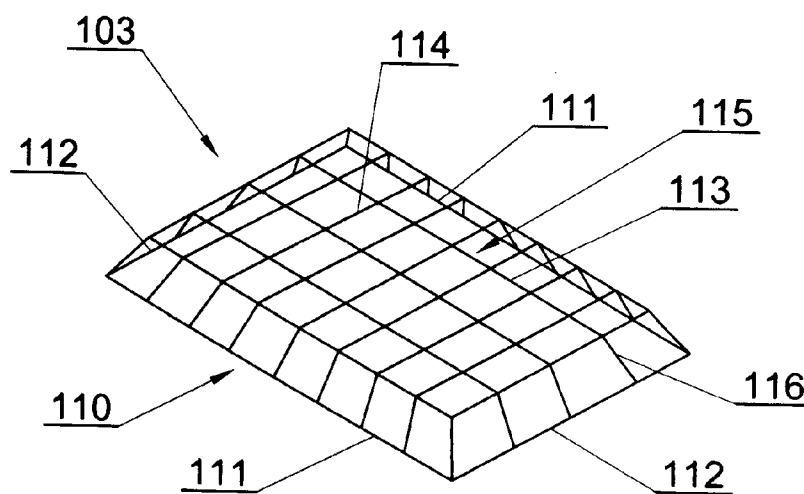


Fig. 2

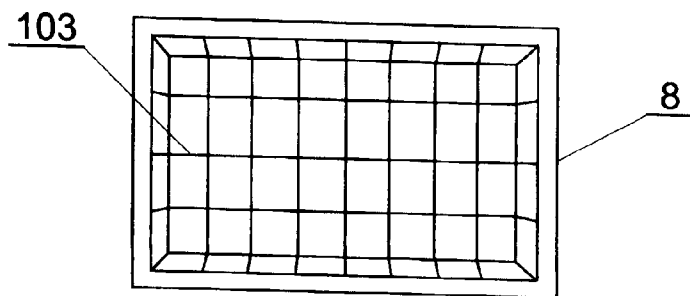


Fig. 3

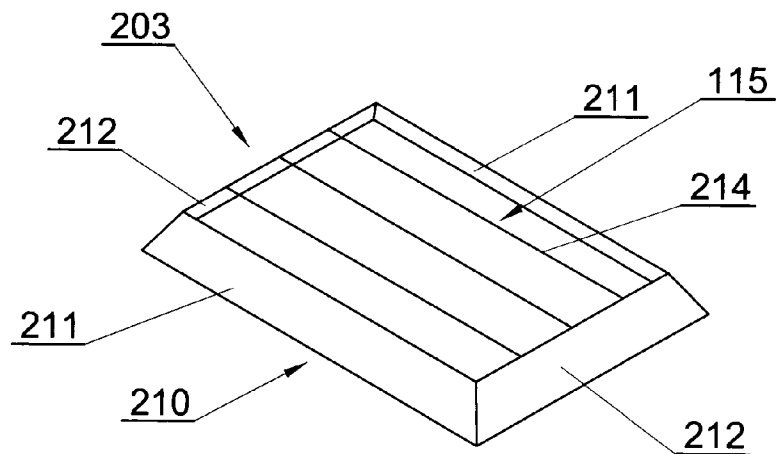


Fig. 4

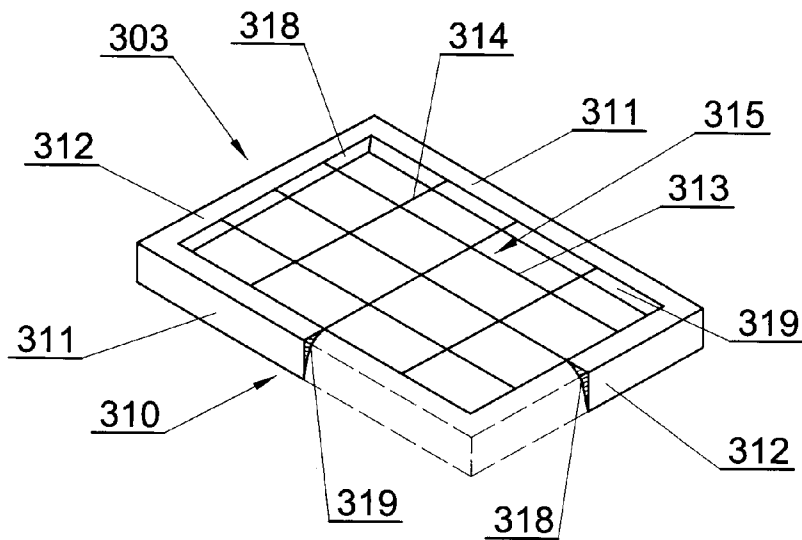


Fig. 5

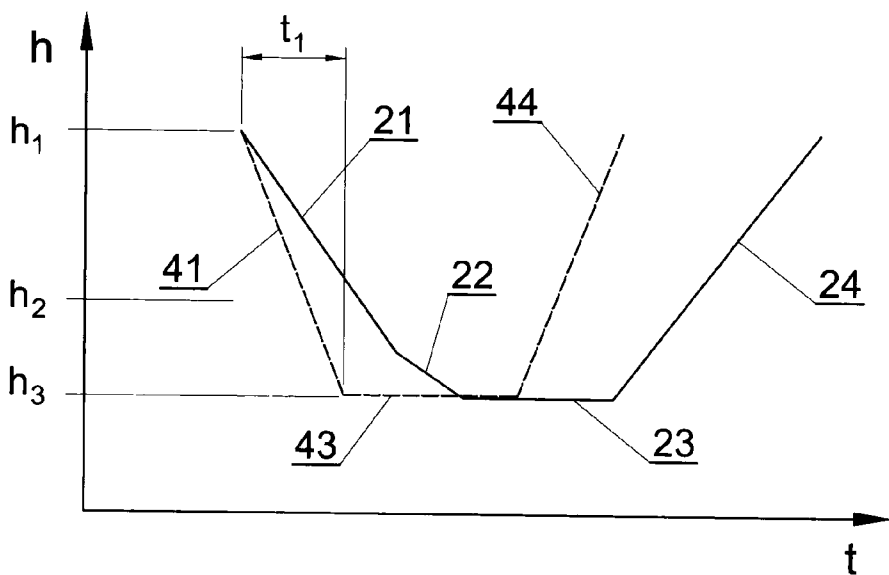


Fig. 6

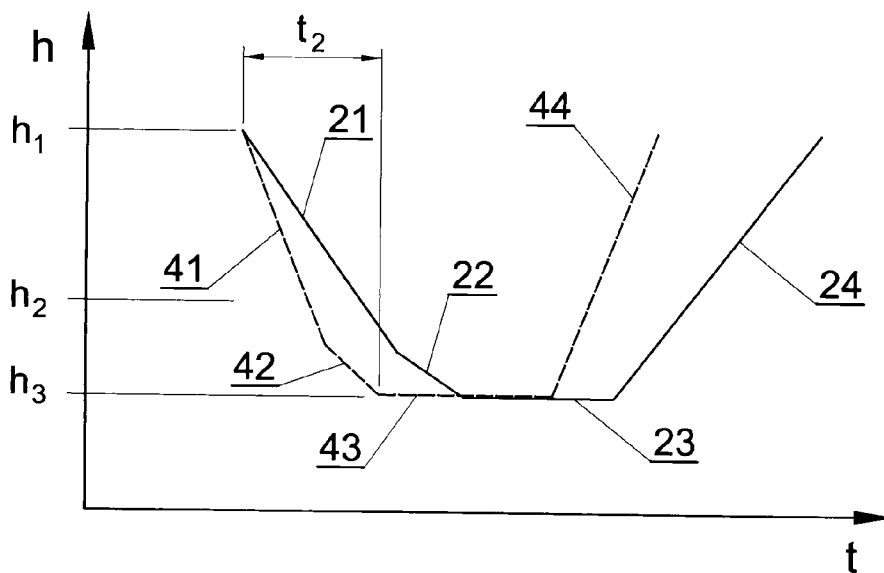


Fig. 7

### ELECTRIC PRESS FOR CUT FILLER COMPRESSION

**[0001]** The object of the invention is an electric press for the compression of cut filler used for cigarette manufacturing.

**[0002]** Cut filler may be stored and transported in boxes, where it is usually compressed so as to utilize the volume of the box to the maximum extent. For cut filler compression, presses with hydraulic and pneumatic drive are commonly used. A high degree of cut filler compression requires the use of actuators with a large working stroke. Hydraulic presses ensure low motion speeds and entail the hazard of product contamination with working liquid, whereas known press design solutions using pneumatic actuators require highly efficient air supply systems, and the control of the motion speed and the piston rod thrust requires complicated control systems. An alternative solution is double compression, where before the second compression additional cut filler feeding takes place, using an actuator with a shorter working stroke.

**[0003]** From the publication U.S. Pat. No. 4,572,065 a press driven by a hydraulic actuator is known in which the compressing element is a heavy metal plate with rectangular shape. The compression takes place in two stages, where the air is sucked off from the compression chamber, and the plate being the compressing element is supposed to assist pressing of cut filler. After the compression, a bale of tobacco or cut filler is pulled out of the compression chamber.

**[0004]** On the other hand, from the publication DE 34 05 182 a solution is known in which cut filler is pressed directly in the box in which it will be stored or transported. The bottom part of a charging hopper is inserted into the box and it will be removed after the compression of cut filler. The part of the charging hopper inserted into the box is provided with an air removal system.

**[0005]** From the publication WO 2005/013732 a cut filler press provided with a flat solid plate driven by four chains with a special construction for applying pressure at the corners of the plate is known.

**[0006]** On the other hand, the publication U.S. Pat. No. 3,782,275 discloses a lightweight design of the compressing element of a press for forming hay bales. The compressing element consists of several elements forming a common compression surface.

**[0007]** In view of the expectation of manufacturers that the pressing process is as short as possible, the motion of the compressing element to reach cut filler should take place at high speed, but in the last phase of the motion the speed should be reduced so as not to cause rising of dust. The effect of rising dust in known solutions using non-openwork compressing elements without reducing the motion speed of these elements is caused by the fact that the air from under the compressing element is squeezed with a considerable speed through the slit around said element formed by the compressing element and the wall of the compression chamber or the box.

**[0008]** Furthermore, while compressing cut filler we have to deal with the impact of cut filler on side walls of the box, which results in bulging of the walls. The effect of stretching the walls of the box is particularly visible in case of uneven filling with cut filler before the compression.

**[0009]** While compressing cut filler, with decreasing layer of cut filler, greater and greater compressing force applied on cut filler by the compressing element is necessary. When

applying constant pressure, the compression process became longer due to the time needed for permanent deformation of tobacco fibres.

**[0010]** A problem that has been solved by the present invention is the optimisation of kinematic characteristics of the compressing element, the optimisation of compression parameters with simultaneous elimination of the effect of bulging walls of the box.

**[0011]** The substance of the invention is a press for the compression of tobacco cut filler provided with at least one electric motor and a control unit driving a movable compressing element by means of a drive train. According to the invention, the compressing element which the press is provided with has an openwork design and is driven while keeping controllable torque.

**[0012]** A press according to the invention is characterised in that the electric motor of the press is a synchronous motor.

**[0013]** A press according to the invention is characterised in that the compressing element is driven while keeping constant driving torque of the motor.

**[0014]** A press according to the invention is characterised in that the drive train includes a driving chain.

**[0015]** A press according to the invention is characterised in that the compressing element has the shape of a frame with transverse bars.

**[0016]** A press according to the invention is characterised in that the dimensions of the compressing element are considerably smaller than the cross-section of a tobacco mass compression chamber.

**[0017]** The solution according to the invention enables the compressing element to make quick motions before the commencement of compression, optimised control of the compression force of the compressing element and quick withdrawal of the compressing element. The structure of the compressing element ensures that the box in which the compression takes place is not deformed.

**[0018]** The object of the invention was shown in preferred embodiments in the drawing in which:

**[0019]** FIG. 1 shows a kinematic diagram of a press,

**[0020]** FIG. 2 shows a compressing element in a first embodiment,

**[0021]** FIG. 3 shows a top view of a box with a compressing element situated inside,

**[0022]** FIG. 4 shows a compressing element in a second embodiment,

**[0023]** FIG. 5 shows a compressing element in a third embodiment,

**[0024]** FIGS. 6 and 7 show performance characteristics of a press with a pneumatic actuator drive and a press according to the present invention,

**[0025]** FIG. 1 shows a kinetic diagram of a press in which the drive is realised by an electric motor 1 controlled by a control unit 2. The drive from the motor 1 is transmitted to a compressing element 3 by a drive train 4 which may be provided with a chain 5 and chain wheels 6. The compressing element can move together with a slider 10 vertically along a guide 7 and sink in a box 8. In the space between the compressing element 3, shown in FIG. 1 in its upper position, and the box 8, conveyor 9 feeding cut filler to the box 8 is situated. In FIG. 1, the conveyor 9 is in a position enabling it to feed cut filler to the box 8.

**[0026]** In order to fill the box 8 with cut filler and to compress it, the box 8 is put under the press so that the compressing element can slip inside the box 8. After putting the box 8,

the conveyor **9** is moved so that cut filler, fed by it, falls inside the box. After feeding a portion of tobacco cut filler, the conveyor **9** is withdrawn in order to make a motion of the compressing element **3** possible.

**[0027]** FIG. **2** shows a compressing element **103** in a first embodiment. A frame **110** of the compressing element consists of two longitudinal elements **111** and two transverse elements **112**. A central raised part **115** of the compressing element **103** is built of longitudinal elements **113** and transverse elements **114**. The central part **115** of the element and the frame **110** are connected by means of diagonally positioned couplers **116**. For reasons of simplification, the elements connecting the compressing element **103** with the slider **10** of the drive train **4** have not been shown.

**[0028]** In the course of compression, cut filler, as a three-dimensional structure in which the fibres catch hold of each other, acts like a sponge. Diagonally positioned couplers **116** belonging to the frame **110** of the compressing element **103** cause that tobacco cut filler is to some extent moved away from the walls of the box. In the course of compression, the air accumulated within tobacco cut filler escapes through the open-work central part **115**.

**[0029]** FIG. **3** shows a top view of the box **8** with the compressing element **103** placed inside. The overall dimensions of the compressing element may be smaller than in case of known solutions where the compressing element fills the compression chamber with a minimum clearance.

**[0030]** FIG. **4** shows a compressing element **203** in a second embodiment. A frame **210** of the compressing element consists of two flat inclined longitudinal elements **211** and two flat inclined transverse elements **212**. A central part **215** of the compressing element is built of longitudinal elements **213** connected with transverse elements **212**.

**[0031]** Diagonally positioned elements **211** and **212** belonging to the frame **210** of the compressing element **203** cause that during compression the tobacco cut filler is to some extent moved away from the walls of the box. During compression, the air accumulated within tobacco cut filler escapes through the openwork central part **215**.

**[0032]** FIG. **5** shows a compressing element **303** in a third embodiment. A frame **310** of the compressing element **303** consists of two longitudinal elements **311** with arched inner surfaces **319** and two transverse elements **312** with arched inner surfaces **318**. A central part **315** of the compressing element is built of longitudinal elements **313** and transverse elements **314** connected with the elements **311** and **312**.

**[0033]** Arched surfaces **318** and **319** of the elements **311** and **312** belonging to the frame **310** of the compressing element **303** cause that during compression the tobacco cut filler is to some extent moved away from the walls of the box. During compression, the air accumulated within tobacco cut filler escapes through the openwork central part **315**.

**[0034]** FIGS. **6** and **7** show a comparison of exemplary simplified performance characteristics of a press with the use of a pneumatic actuator and a press according to the present invention. The vertical axis of the diagram shows the position of the compressing element in relation to the bottom of the compression chamber or the box, the horizontal axis is a time axis. The characteristics relates to a shift of the compressing element from a height  $h_1$  corresponding to an upper position of the compressing element to a height  $h_2$  corresponding to the height of a compressed bale of cut filler and the return of the compressing element to the initial position, i.e. to the height  $h_1$ . The height of loosely filled cut filler before com-

pression has been marked with  $h_3$ . The compression process may be divided into three phases:

**[0035]** a pushing the compressing element close to cut filler and compression of cut filler,

**[0036]** b keeping cut filler in compressed state until the moment when cut filler loses resilience, i.e. the fibres of cut filler will be permanently deformed, so that the upper surface of the bale of compressed cut filler will not rise after withdrawing the compressing element,

**[0037]** c withdrawal of the compressing element.

**[0038]** The section **21** (FIG. **6**) of the diagram shows a shift of the compressing element driven by a pneumatic actuator including pushing the compressing element close to cut filler and the commencement of compression, and the section **22** shows further shifting of the compressing element until the moment when the resistance of partly compressed cut filler increases; fragments **21** and **22** correspond to the phase a. The section **23** corresponds to the phase b, and the section **24** to the phase c of the compression process. The section **41** of the diagram shows a shift of the compressing element driven by an electric motor. The motor is controlled by the control unit in such a way as to move the compressing element with a constant speed, i.e. its driving torque, thus the compression force of the compressing element, are adjusted to growing resistance of compressed cut filler. The section **41** corresponds to the phase a, the section **43** to the phase b, whereas the section **44** to the phase c of the compression process.

**[0039]** FIG. **7** shows sections **21**, **22**, **23** and **24** describing a shift of the compressing element driven by a pneumatic actuator as in FIG. **6** as well as sections **41**, **42**, **43** and **44** describing a shift of the compressing element driven by a press according to the present invention. The section **41** relating to the first part of the phase a, shows a shift of the compressing element including pushing the element close to cut filler and the commencement of compression until the moment when the resistance of compression is lower than the force which may be applied by the compressing element. The section **42** shows a shift of the compressing element having impact on cut filler with a constant force resulting from the applied constant driving torque of the motor. The time  $t_2$  necessary for the compression of cut filler to the height  $h_2$  is longer than the time  $t_1$  in case of shifting the compressing element with a constant speed, nevertheless, the total time of the compression process is shorter than in case of compression by means of a press driven by a pneumatic actuator.

**1-6.** (canceled)

**7.** An electric press for the compression of tobacco cut filler provided with

at least one electric motor and a control unit driving a movable compressing element by means of a drive train characterized in that the compressing element has an open-work design comprising flat, inclined longitudinal elements and is driven while keeping controllable driving torque.

**8.** The press according to claim **7** wherein the electric motor is a synchronous motor.

**9.** The press according to claim **7** wherein the compressing element is driven while keeping a constant driving torque of the motor.

**10.** The press according to claim **8** wherein the compressing element is driven while keeping a constant driving torque of the motor.

**11.** The press according to claim **7** wherein the drive train includes a driving chain.

**12.** The press according to claim **8** wherein the drive train includes a driving chain.

**13.** The press according to claim **9** wherein the drive train includes a driving chain.

**14.** The press according to claim **20** wherein the drive train includes a driving chain.

**15.** The press according to claim **7** wherein the compressing element has the shape of a frame with transverse bars.

**16.** The press according to claim **8** wherein the compressing element has the shape of a frame with transverse bars.

**17.** The press according to claim **9** wherein the compressing element has the shape of a frame with transverse bars.

**18.** The press according to claim **10** wherein the compressing element has the shape of a frame with transverse bars.

**19.** The press according to claim **15** wherein the dimensions of the compressing element are considerably smaller than the cross-section of the tobacco mass compression chamber.

**20.** The press according to claim **16** wherein the dimensions of the compressing element are considerably smaller than the cross-section of the tobacco mass compression chamber.

**21.** The press according to claim **17** wherein the dimensions of the compressing element are considerably smaller than the cross-section of the tobacco mass compression chamber.

**22.** The press according to claim **18** wherein the dimensions of the compressing element are considerably smaller than the cross-section of the tobacco mass compression chamber.

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